# Single Point Mooring (SPM) Systems with Buoys

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The SPM system consists of four main components, namely, the body of the buoy, the anchoring and mooring components, the fluid transfer system and the ancillary elements. Static legs linked to the seabed underneath the surface keep the buoy body in place. Above the water level, the body has a spinning portion that is attached to the offloading/loading tanker. A roller bearing, referred to as the main bearing, connects these two portions. Due to this array, the anchored tanker can easily weather-vane around the buoy and find a steady position. The concept of the buoy is determined by the type of bearing utilized and the divide between the rotating and geostatic sections. The buoy's size is determined by the amount of counter buoyancy required to keep the anchor chains in place, and the chains are determined by environmental conditions and vessel size.

Keywords: marine hose model ; Catenary Anchor Leg Moorings (CALM) buoy ; ocean waves

### 1. Categorisation of SPM Moorings

CALM buoy is an application of SPM mooring systems <sup>[1][2][3][4][5][6][7][8][9][10]</sup>. There are three categories of SPM moorings that will be looked: SPMs, CALM buoys and marine hose systems. These are based on their operational relevance to the CALM buoy system or the connecting FPSO tanker in an SPM unloading or discharging hose system, like deepwater lines, Oil Offloading Lines (OLLs), flexible riser pipes, flexible hoses, and other CALM buoy hose systems <sup>[11][12][13][14][15]</sup> <sup>[16][17][18][19][20]</sup>. To avoid failure, safety must be key for installation and (un)loading <sup>[21][22][23]</sup>. An operation to replace or install components can be carried out to change the complete buoy hose system, like on the SBM buoy in Djeno Teminal, Congo <sup>[24]</sup>, as depicted in **Figure 1**.

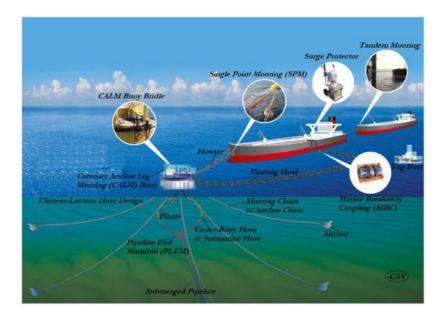


**Figure 1.** Full replacement operation of floating hoses, submarine hoses, hawsers and a single point mooring (SPM) buoy attached to a service offshore vessel (SOV) by a tug supply boat, located at 35 m water depth in Republic of Congo, Djeno Terminal (Courtesy: Bluewater & South Offshore <sup>[24]</sup>).

Once the floating buoy is secured to the seafloor by moorings, next is the anchoring systems. These might be made up of ships, rigs, piles, or gravity anchors, depending on the local soil conditions. Based on the SPM classification, CALM and SALM are the two most common mooring systems for SPMs. In a CALM system, the buoy is held in place by the CALM's anchor chain, which runs in catenaries towards anchor points slightly further away from the buoy. The SALM system is similar, with the exception that the SALM is only anchored by one anchor leg. The key advantage of a CALM buoy over a SALM buoy is it is easy to maintain. CALM buoys have been deployed in the vast majority of Marine Terminals since the mid-1990s.

## 2. Components of SPM System

Generally, the mooring lines, connectors, and anchors make up a mooring system. The mooring wires can also be used to connect buoys and clump weights. A mooring line can be made of a variety of materials, such as chains, fiber ropes, or wire ropes. **Figure 2** represents a CALM buoy system, three mooring configurations and various components, adapted from <sup>[25]</sup>. The three mooring configurations seen on **Figure 2** are the Chinese lantern configuration, single point mooring (SPM), and tandem mooring.



**Figure 2.** Catenary Anchor Leg Mooring (CALM) buoy hose system showing Chinese lantern configuration, single point mooring (SPM), and tandem mooring. It shows the Marine Breakaway Coupling (MBC), anchor, mooring chain or anchor chain, floating hose, under-buoy hose or submarine hose, buoyancy floats, CALM buoy, hawsers, surge protector, tug boat, submerged pipeline, pipeline end manifold (PLEM) and the CALM buoy bridle. (Adapted with permission <sup>[25]</sup>).

**Figure 2** also shows various components like the Marine Breakaway Coupling (MBC), floating hose, under-buoy hose or submarine hose, buoyancy floats, CALM buoy, surge protector, tug boat, submerged pipeline, pipeline end manifold (PLEM), hawsers, CALM buoy bridle, anchor, and anchor chain or mooring chain.

Harnois <sup>[26][27]</sup> provided a comparison of various mooring line materials. The inertia, elastic stiffness, and damping of a mooring line are affected by the material used. An anchor's purpose is to secure a mooring line to a fixed place on the bottom. The ability to resist high, horizontal, and in some cases vertical loads in a specific seabed type (soft to hard), cost-effectiveness, and ease of installation are the major requirements for an anchor. There are several types of anchors available, including dead weight, drag embedment, pile anchor, and plate anchor. It is noteworthy to add that the use of hawser is dependent on the size of the vessel to be anchored to the buoy, as hawser systems can use one or two ropes, as depicted in **Figure 1** and **Figure 2**.

A mooring system is made up of many materials and components that are organized in a specific way, as shown in **Figure 3**. The other SPM components are as follows:

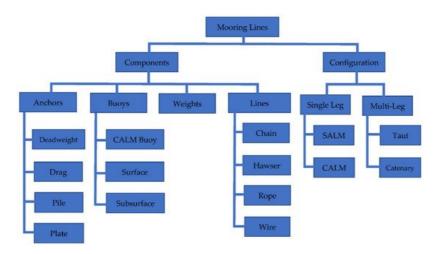


Figure 3. Illustration of the components and configurations for a mooring system. [Illustration design: by Author1].

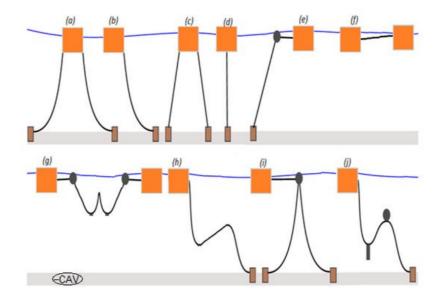
- · The access to the buoy deck is provided by a boat landing;
- · The buoy is protected by fenders;
- · The material handling equipment includes lifting and handling equipment;
- Maritime visibility aids and a fog siren are used to keep moving vessels alert and attentive;
- The navigation aids or other equipment are powered by the electrical subsystem;
- The sources of power systems are batteries and solar systems. While the batteries are replenished on a regular basis, the solar power systems employ sun-sourced renewable energy and maintain the charge in the battery packs, for electrical power;
- A hydraulic system can be added for remote operation with PLEM valves, if needed.

#### 3. Components of CALM Buoy System

The Catenary Anchor Leg Mooring (CALM) buoy system has a buoy with a pivot, called the turntable. This rotates around the vertical axis of the pivot, as the tanker is moored to it. The floating hose is also connected to the turntable, at an angle through the hose manifold. The elastically moored buoy of radius a is acted upon by a wave train of irregular waves and wave height H progressing in x-direction, as illustrated in **Figure 1**. The turntable on the mooring buoy can spin around its vertical axis. The tanker is moored to the turntable and is connected to the floating hose strings that are also attached to the turntable. Due to the forces imposed by the currents and waves, the entire system can freely rotate, which is termed weather-vaning. **Figure 1** and **Figure 3** show Catenary Anchor Leg Mooring (CALM) buoy systems. Basically, there are three CALM system mooring components, namely, the anchors, the chain anchors and the chain stoppers. The anchors are used to hold things together, including the piles or gravity anchors for connecting the seabed with the mooring chain. The most common chain anchors are systems with either six or eight anchor chains. The third component is the stoppers for chains, which are for connecting the buoy with the mooring chains. The anchor chains help to keep the buoy in place. The fluid is transferred to the submarine hose strings via a swivel, which links to the undersea pipeline via the pipeline end manifold.

#### 4. Different Mooring Configuration

There are other types of offshore mooring systems, aside SPM. Based on the mathematical modeling, HMs and MMs, considering SPMs for bonded marine hoses, have been developed over 45 years based on earlier works on point moorings and simple floating buoys. The application of offshore hoses has also led to advances in different mooring systems used in fluid transfer, as seen in **Figure 4**.

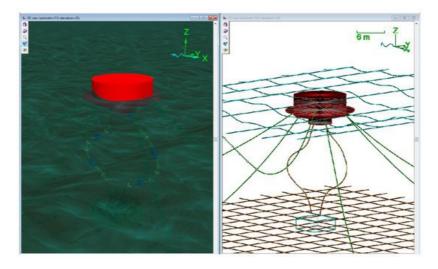


**Figure 4.** Configurations for mooring lines showing: (a) multi-catenary taut; (b) catenary; (c) taut; (d) spread; (e) SALM; (f) ship-to-ship catenary; (g) weight-added connection; (h) Lazy-S, (i) CALM; and (j) Steep-S. [Sketch design: by Author1].

Moorings are also applied on shipping vessels for other oil field operations like CO<sub>2</sub> oil recovery <sup>[28]</sup>. Several studies assessed mooring statics and dynamics for CALM buoy, as well as with attached hoses, which were considered as a single point mooring (SPM) terminal <sup>[29][30][31][32][33]</sup>. The design of each hose-mooring system considers different loadings, predictive motion responses with structural statics/dynamics <sup>[34][35][36][37][38][39][40][41][42][43][44]</sup>, and governing theories on the hydrodynamics of floating structures <sup>[45][46][47][48][49][50][51][52][53][54]</sup>. In addition, the design of FOS is based on different industry standards <sup>[55][56][57][58][59]</sup>. The application of a mooring configuration is based on the application requirement, the type of (un)loading operation, and the environmental conditions. Some of these mooring applications require floating, catenary, and reeling hoses, while others require submarine hoses.

#### 5. Review on Physical Models on Hoses and SPMs

The selection of hose systems for single point mooring (SPM) systems has been described by Ziccardi and Robbins <sup>[60]</sup>. Setting up buoys in low-tide areas was discussed as the authors also wanted to stimulate more hose and flexible rubber pipeline designs and applications. They studied the SPM deployments at Tokyo Bay's Hakozaki and Koshiba terminals. They also included a timeline of hose design and trends. They claimed that the basic designs of under-buoy hoses and floating hoses are comparable. The strong crush resistance of sub-surface hoses, on the other hand, was shown to be dependent on the water depth. This was accomplished by increasing either the wire's area or the diameter of the helical wire, or both. The rated operating pressure was found to be 5 to 6 times the design burst pressure. They highlighted the abrasion and abuse that the floating hoses attached to the tanker from the buoy were subjected to. They came to the conclusion that developing flexible rubber lines that could sustain high operating pressures and external crush, particularly in severe environments, was critical. The hose system for an SPM terminal was also reliant on both the operational and environmental conditions, according to the report. Physical tests are also used to develop environmental wave spectra, such as the Joint North Sea Wave Project (JONSWAP) wave spectrum and regular wave types like Airy waves <sup>[61][62][63]</sup>. Typical recent numerical model of CALM buoy model conducted in Orcina's Orcaflex by the authors can be seen in **Figure 5**.



**Figure 5.** CALM buoy model using Chinese lantern configuration under an ocean environment in Orcaflex 11.0f, showing a shaded view and a wireframe view. [Model design: by Author1].

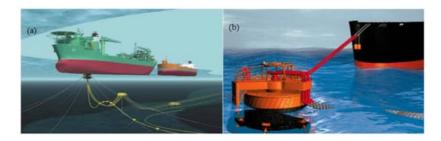
The operational requirements, such as the system's working pressure, necessitated the transportation of well-specified products with an adequately defined nature. They listed several factors that must be considered when determining the length of hose strings in hose designs, including mean water depth, tide depth (low/high), maximum wave height, buoy position relative to pipeline header, maximum mooring distance, rated working pressure, desired throughput, and product(s) to be transported. They advised that the ultimate design of the under-buoy system should ensure that the hose does not come into direct contact with the seabed of the moored ship under high tide conditions. They looked at the primary design criteria for SPM hoses, underbuoy system, floating hose systems, float sinks, hose designs, and hose diameters, and encouraged greater research from hose manufacturers, using the two case studies that were employed by the US military on unloading from SPM tankers.

Earlier investigations on marine hoses depended on some lengthy calculations and experiments. Brady et al. <sup>[66]</sup> with the help of Shell B.P Petroleum Company of Nigeria Limited, built a test apparatus that was connected to 60.96 cm (24 inches) hoses attached to a CALM buoy off the coast of Nigeria. A Medilog 4–24 small four-channel cassette recorder, a 4–366 pressure transducer, and a Beaulieu S.P 16 mm Cine camera with a lens width of 5.9 mm were also included in the setup. To measure the strains on the hose of a monobuoy, a strain-gauge measuring spool was installed between the

buoy manifold and the first-off buoy hose. They claimed that the hoses closest to the buoy have a lower life expectancy because they carry the majority of the hose stresses. Correlation of the measured loads was achievable using the statistical method described for calculating the 60 s recordings and visual records of the sea conditions. However, this was limited due to a lack of environmental data. Rather than the trial-and-error method employed previously, this technology enabled the investigation of the forces on buoy hoses. They came to the conclusion that the hose problem was primarily caused by fatigue rather than high loads. As a result, increasing the hoses' strength will improve their performance. SPM terminals were subjected to model testing by Pinkster and Remery [67]. The test results were also used to describe SPM terminal features and hose phenomena found in CALM and SALM mooring systems. They also stressed the need for selecting the appropriate scale for model tests. They cited water depth, the accuracy of the results, and the capacity to generate the needed wave height and period at a certain scale in the basin as critical variables. Water depth, current, wave generators, and wind are some of the variables that can be modified to affect environmental conditions. They also went over the model testing technique, measurements, and analysis in detail. They concluded that nonlinearities were exploited in the construction of the equation of motion, which was then integrated in small time increments step by step. Additionally, based on uncertainties in the prototype's drag coefficients, the inaccuracies in the estimates of the findings obtained from the model tests due to scale effects should be applied without modification. There was additional discussion of the Pierson-Moskowitz spectrum, wind forces, current forces, first-order wave forces, second-order wave forces, and drag wave drifting equations. However, the methodologies for calculating these forces were not sufficiently developed for design consistency. An industry collaboration with academia was conducted on the feasibility of using geodesic IGW designs for offloading hoses, as reported in Nooij [68]. Another important study that was carried out on the load response of offshore hoses by Lassen T. et al. [69] involved finite element models and full-scale testing for a 20inches-bonded hose with steel end fittings. The study presents limits based on API 17K <sup>[70]</sup> criteria for the extreme load capacity assessments. The study also included a methodology for predicting the fatigue life of bonded loading hoses' response to applied bending, tension and pressure using a catenary configuration, with reeling loadings repeated and significantly tensioned. The study emphasized the fatigue life prediction methods, as well as the load impacts on the hose during reeling operations, for both rubber and steel parts.

#### 6. FPSOs for Marine Hose Operations

There are different types of FPSOs that are used in SPMs for transfer, loading and offloading operations. The turret systems are the most common because of their freedom of movement, ease of anchorage, and accessibility during mooring and deployment. A typical turret FPSO in catenary mooring is shown in **Figure 6**a, and an Offloading FPSO attached to an SPM's CALM buoy is shown in **Figure 6**b. A variety of numerical models on other mooring systems can be seen in the literature and existing industry projects on marine hose, as earlier discussed.



**Figure 6.** Typical Floating Production Storage and Offloading (FPSO) systems showing (**a**) a turret FPSO with catenary moorings and (**b**) an Offloading FPSO attached to a CALM buoy using single point mooring by 2 hawsers and 3 floating hoses.

#### References

- 1. Trelleborg. Surface Buoyancy. Trelleborg Marine and Infrastructure: Product Brochure. 2017. Available online: https://w ww.trelleborg.com/en/marine-and-infrastructure/products-solutions-and-services/marine/surface-buoyancy (accessed o n 17 September 2021).
- ContiTech. Marine Hose Brochure. 2020. Available online: https://aosoffshore.com/wp-content/uploads/2020/02/ContiTe ch\_Marine-Brochure.pdf (accessed on 17 February 2021).
- 3. ContiTech. Marine Hoses-Offshore Fluid Transfer; Continental Dunlop; Contitech Oil & Gas: Grimsby, UK, 2017; Availa ble online: http://www.contitech-oil-gas.com/pages/marine-hoses/marine-hoses\_en.html (accessed on 17 September 2 021).

- 4. ContiTech. High Performance Flexible Hoses Brochure; Continental Dunlop; Contitech Oil & Gas: Grimsby, UK, 2014.
- 5. Alfagomma. Industrial Hose & Fittings; Alphagomma SpA: Vimercate, Italy, 2016.
- 6. SBMO. SBMO CALM Brochure; SBM Offshore: Amsterdam, The Netherlands, 2012.
- Technip. Coflexip® Flexible Steel Pipes for Drilling and Service Applications: User's Guide; Technip: Paris, France, 200
  6.
- Trelleborg. Trelline Catalogue. Trelleborg, France. 2014. Available online: http://www2.trelleborg.com/Global/WorldOfTr elleborg/Fluid%20handling/TRELLINE%20Catalogue.pdf. (accessed on 17 May 2021).
- OIL. Offloading Hoses: Floating & Submarine Hoses-OIL Hoses Brochure; Offspring International Limited: Dudley, UK, 2014; Available online: https://www.offspringinternational.com/wp-content/uploads/2020/06/OIL-Offloading-Hoses-Broc hure-2020-W.pdf (accessed on 12 July 2021).
- 10. Maslin, E. Unmanned Buoy Concepts Grow; Offshore Engineer: New York, NY, USA, 2014; Volume 1, Available online: http://www.oedigital.com/component/k2/item/5621-unmanned-buoy-concepts-grow. (accessed on 17 May 2021).
- 11. Xu, X.; Chen, G.; Zhu, W.; Shi, Y.; Gao, W.; Jin, W. Study on a New Concept of Offloading System for SDPSO. In Proce edings of the 30th International Ocean and Polar Engineering Conference, Shanghai, China, 11–16 October 2020.
- Oliveira, M.C. Ultradeepwater Monobuoys, OMAE2003-37103. In Proceedings of the International Conference on Offsh ore Mechanics & Arctic Engineering, Cancun, Mexico, 8–13 June 2003; pp. 1–10.
- Quash, J.E.; Burgess, S. Improving Underbuoy Hose System Design Using Relaxed Storm Design Criteria. In Proceedings of the Offshore Technology Conference, Houston, TX, USA, 30 April 30–3 May 1979.
- Carpenter, E.B.; Idris, K.; Leonard, J.W.; Yim, S.C.S. Behaviour of a moored Discus Buoy in an Ochi-Hubble Wave Spe ctrum. In Proceedings of the Offshore Technology Conference Proceeding, Houston, TX, USA, 27 February–3 March 1 994; pp. 347–354. Available online: http://web.engr.oregonstate.edu/~yims/publications/OMAE1994.DiscusBuoy.pdf (ac cessed on 17 June 2021).
- Rampi, L.; Lavagna, P.; Mayau, D. TRELLINE? A Cost-Effective Alternative for Oil Offloading Lines (OOLs). Paper Num ber: OTC-18065-MS. In Proceedings of the Paper Presented at the Offshore Technology Conference, Houston, TX, US A, 1–4 May 2006.
- 16. Prischi, N.; Mazuet, F.; Frichou, A.; Lagarrigue, V. SS-Offshore Offloading Systems and Operations Bonded Flexible Oil Offloading Lines, A Cost Effective Alternative to Traditional Oil Offloading lines. Paper Number: OTC-23617-MS. In Proc eedings of the Paper Presented at the Offshore Technology Conference, Houston, TX, USA, 30 April–3 May 2012.
- Mayau, D.; Rampi, L. Trelline—A New Flexible Deepwater Offloading Line (OLL). In Proceedings of the 16th Internation al Offshore and Polar Engineering Conference, San Francisco, CA, USA, 28 May–2 June 2006; Available online: http s://onepetro.org/ISOPEIOPEC/proceedings-abstract/ISOPE06/All-ISOPE06/ISOPE-I-06-127/9875 (accessed on 17 Ju ne 2021).
- Tschoepe, E.C.; Wolfe, G.K. SPM Hose Test Program. In Proceedings of the Offshore Technology, Houston, TX, USA, 4–7 May 1981; pp. 71–80.
- 19. Zhang, S.-F.; Chen, C.; Zhang, Q.-X.; Zhang, N.-M.; Zhang, F. Wave Loads Computation for Offshore Floating Hose Ba sed on Partially Immersed Cylinder Model of Improved Morison Formula. Open Pet. Eng. J. 2015, 8, 130–137.
- Lebon, L.; Remery, J. Bonga: Oil Off-loading System using Flexible Pipe. In Proceedings of the Offshore Technology C onference Proceeding-OTC 14307, Houston, TX, USA, 6 May 2002; pp. 1–12.
- 21. Asmara, I.P.S.; Wibowo, V.A.P. Safety Analysis of Mooring Hawser of FSO and SPM Buoy in Irregular Waves. In Proce edings of the Maritime Safety International Conference. IOP Conf. Ser. Earth Environ. Sci. 2020, 557, 012003.
- 22. Løtveit, S.A.; Muren, J.; Nilsen-Aas, C. Bonded Flexibles–State of the Art Bonded Flexible Pipes; 26583U-1161480945 -354, Revision 2.0, Approved on 17.12.2018; PSA: Asker, Norway, 2018; pp. 1–75. Available online: https://www.4subs ea.com/wp-content/uploads/2019/01/PSA-Norway-State-of-the-art-Bonded-Flexible-Pipes-2018\_4Subsea.pdf (accesse d on 17 June 2021).
- 23. Muren, J.; Caveny, K.; Eriksen, M.; Viko, N.G.; MÜLler-Allers, J.; JØRgen, K.U. Un-Bonded Flexible Risers–Recent Fiel d Experience and Actions for Increased Robustness; 0389-26583-U-0032, Revision 5.0; PSA: Asker, Norway, 2013; Vol ume 2, pp. 1–78. Available online: https://www.ptil.no/contentassets/c2a5bd00e8214411ad5c4966009d6ade/un-bonde d-flexible-risers--recent-field-experience-and-actions--for-increased-robustness.pdf (accessed on 17 June 2021).
- 24. SouthOffshore. Congo BW CALM Buoy Full Replacement with New SBM Buoy Project. Djeno Terminal. South Offshor e. 2018. Available online: https://www.south-offshore.com/portfolio/djeno-terminal/ (accessed on 17 May 2021).
- 25. OIL. Mooring and Offloading Systems; Offspring International Limited: Dudley, UK, 2015; Available online: https://www. offspringinternational.com/wp-content/uploads/2015/04/OIL-SPM-Brochure-2015.pdf (accessed on 12 July 2021).

- 26. Harnois, V. Analysis of Highly Dynamic Mooring Systems: Peak Mooring Loads in Realistic Sea Conditions. Ph.D. Thes is, University of Exeter, Exeter, UK, 2014.
- 27. Harnois, V.; Weller, S.D.; Johanning, L.; Thies, P.R.; Le Boulluec, M.; Le Roux, D.; Soule, V.; Ohana, J. Numerical mod el validation for mooring systems: Method and application for wave energy converters. Renew. Energy 2015, 75, 869–8 87.
- Brownsort, P. Offshore offloading of CO2: Review of Single Point Mooring Types and Suitability. In SCCS (Scottish Car bon Capture and Storage); SCCS CO2-Enhanced Oil Recovery Joint Industry Project: Eaton Socon, UK, 2015; pp. 1–2
   Available online: https://www.sccs.org.uk/images/expertise/misc/SCCS-CO2-EOR-JIP-Offshore-offloading.pdf (acces sed on 19 July 2021).
- 29. Gao, Z.; Moan, T. Mooring system analysis of multiple wave energy converters in a farm configuration. In Proceedings of the 8th European Wave and Tidal Energy Conference, Uppsala, Sweden, 7–10 September 2009; pp. 509–518.
- 30. Sound and Sea Technology. Advanced Anchoring and Mooring Study; Technical Report; Oregon Wave Energy Trust: P ortland, OR, USA, 2009.
- 31. Angelelli, E.; Zanuttigh, B.; Martinelli, L.; Ferri, F. Physical and numerical modelling of mooring forces and displacement s of a Wave Activated Body Energy Converter. In Proceedings of the ASME 2014 33rd International Conference on Oc ean, Offshore and Arctic Engineering, San Francisco, CA, USA, 8–13 June 2014; p. V09AT09A044.
- 32. Petrone, C.; Oliveto, N.D.; Sivaselvan, M.V. Dynamic Analysis of Mooring Cables with Application to Floating Offshore Wind Turbines. J. Eng. Mech. 2015, 142, 1–12.
- 33. Wichers, I.J. Guide to Single Point Moorings; WMooring Inc: Houston, TX, USA, 2013; Available online: http://www.wm ooring.com/files/Guide\_to\_Single\_Point\_Moorings.pdf (accessed on 17 June 2021).
- 34. Bishop, R.E.D.; Price, W.G. Hydroelasticity of Ships; Cambridge University Press: New York, NY, USA, 2005.
- 35. Bruschi, R.; Vitali, L.; Marchionni, L.; Parrella, A.; Mancini, A. Pipe technology and installation equipment for frontier de ep water projects. Ocean Eng. 2015, 108, 369–392.
- 36. Brebbia, C.A.; Walker, S. Dynamic Analysis of Offshore Structures, 1st ed.; Newnes-Butterworth & Co. Publishers Ltd: London, UK, 1979.
- 37. Chandrasekaran, S. Dynamic Analysis and Design of Offshore Structures, 1st ed.; Springer: New Delhi, India, 2015.
- 38. Bai, Y.; Bai, Q. Subsea Pipelines and Risers, 1st ed.; 2013 RePrint; Elsevier: Oxford, UK, 2005.
- Raheem, S.E.A. Nonlinear response of fixed jacket offshore platform under structural and wave loads. Coupled Syst. M ech. Int. J. 2013, 2, 111–126.
- 40. Papusha, A.N. Beam Theory for Subsea Pipelines: Analysis and Practical Applications; Wiley-Scrivener: Hoboken, NJ, USA, 2015.
- 41. Barltrop, N.D.P.; Adams, A.J. Dynamics of Fixed Marine Structures, 3rd ed.; Butterworth Heinemann: Oxford, UK, 1991.
- 42. Barltrop, N.D.P. Floating Structures: A Guide for Design and Analysis; Oilfield Publications Limited (OPL): Herefordshir e, UK, 1998; Volume 1.
- 43. Wilson, J.F. Dynamics of Offshore Structures, 2nd ed.; John Wiley and Sons: New York, NY, USA, 2003.
- 44. Bai, Y.; Bai, Q. Subsea Engineering Handbook, 1st ed.; Elsevier: Oxford, UK, 2012.
- 45. Newman, J.N. Marine Hydrodynamics; 1999 Repri; IT Press: London, UK, 1977.
- 46. Chakrabarti, S.K. Handbook of Offshore Engineering; Elsevier: Oxford, UK, 2005; Volume 1.
- 47. Chakrabarti, S.K. Hydrodynamics of Offshore Structures; Reprint; WIT Press: Southampton, UK, 2001.
- Chakrabarti, S.K. Offshore Structure Modeling-Advanced Series on Ocean Engineering; World Scientific: Singapore, 19 94; Volume 9.
- 49. Chakrabarti, S.K. The Theory and Practice of Hydrodynamics and Vibration-Advanced Series on Ocean Engineering; World Scientific: Singapore, 2002; Volume 20.
- 50. Chakrabarti, S.K. Handbook of Offshore Engineering; Elsevier: Oxford, UK, 2006; Volume 2.
- Faltinsen, O.M. Sea Loads on Ships and Offshore Structures; 1995 Repri; Cambridge University Press: Cambridge, U K, 1990.
- Païdoussis, M.P. Fluid-Structure Interactions: Slender Structures and Axial Flow, 2nd ed.; Elsevier Ltd: Oxford, UK, 201
  4.

- 53. Morison, J.R.; O'Brien, M.P.; Johnson, J.W.; Schaaf, S.A. The Force Exerted by Surface Waves on Piles. J. Pet. Techn ol. (Pet. Trans. AIME) 1950, 2, 149–154.
- 54. Walker, B. Dynamic Analysis of Offshore Structures; Newnes-Butterworths: London, UK, 2013.
- 55. ABS. Rules For Building And Classing-Single Point Moorings; American Bureau of Shipping: New York, NY, USA, 2021. Available online: https://ww2.eagle.org/content/dam/eagle/rules-and-guides/current/offshore/8\_rules-forbuildingandclas singsinglepointmoorings\_2021/spm-rules-jan21.pdf (accessed on 17 August 2021).
- 56. DNV. Offshore Standard: Dynamic Risers DNV-OS-F201; Det Norske Veritas: Oslo, Norway, 2010.
- 57. DNVGL. DNVGL-OS-E403 Offshore Loading Buoys; Det Norske Veritas & Germanischer Lloyd: Oslo, Norway, 2015.
- 58. DNVGL. DNVGL-RP-N103 Modelling and Analysis of Marine Operations; Det Norske Veritas & Germanischer Lloyd: O slo, Norway, 2017.
- 59. DNVGL. DNVGL-RP-F205 Global Performance Analysis of Deepwater Floating Structures; Det Norske Veritas & Germ anischer Lloyd: Oslo, Norway, 2017.
- 60. Ziccardi, J.J.; Robbins, H.J. Selection of Hose Systems for SPM Tanker Terminals. In Proceedings of the Offshore Tech nology Conference Proceeding-OTC 1152, Dallas, TX, USA, 21 April 1970; pp. 83–94.
- 61. Hasselmann, K.; Barnett, T.P.; Bouws, E.; Carlson, H.; Cartwright, D.E.; Enke, K.; Ewing, J.A.; Gienapp, H.; Hasselman n, D.E.; Kruseman, P.; et al. Measurements of wind-wave growth and swell decay during the Joint North Sea Wave Proj ect (JONSWAP). In Ergänzungsheft zur Dtsch. Hydrogr. Z. -Hydraulic Engineering Reports; Deutches Hydrographische s Institut: Hamburg, Germany, 1973; pp. 1–90. Available online: http://resolver.tudelft.nl/uuid:f204e188-13b9-49d8-a6dc -4fb7c20562fc (accessed on 4 March 2021).
- 62. Chakrabarti, S.K. Technical Note: On the formulation of Jonswap spectrum. Appl. Ocean Res. 1984, 6, 175–176.
- 63. Isherwood, R. Technical note: A revised parameterisation of the Jonswap spectrum. Appl. Ocean Res. 1987, 9, 47–50.
- 64. Chibueze, N.O.; Ossia, C.V.; Okoli, J.U. On the Fatigue of Steel Catenary Risers. J. Mech. Eng. 2016, 62, 751–756.
- Pierson, W.J.; Moskowitz, L. A proposed spectral form for fully developed wind seas based on the similarity theory of S. A. Kitaigorodskii. J. Geophys. Res. Space Phys. 1964, 69, 5181–5190.
- Brady, I.; Williams, S.; Golby, P. A study of the Forces Acting on Hoses at a Monobuoy Due to Environmental Condition s. In Proceedings of the Offshore Technology Conference Proceeding-OTC 2136, Dallas, TX, USA, 5–7 May 1974; pp. 1–10.
- 67. Pinkster, J.A.; Remery, G.F.M. The role of Model Tests in the design of Single Point Mooring Terminals. In Proceedings of the Offshore Technology Conference Proceeding-OTC 2212, Dallas, TX, USA, 4–7 May 1975; pp. 679–702.
- 68. Nooij, S. Feasibility of IGW Technology in Offloading Hoses. Masters Dissertation; Civil Engineering Department: Delft, The Netherlands, 2006; Available online: http://resolver.tudelft.nl/uuid:4617e7a0-b5d8-4c86-94d5-8d2037b31769 (acce ssed on 19 July 2021).
- Lassen, T.; Lem, A.I.; Imingen, G. Load response and finite element modelling of bonded offshore loading hoses. In Pro ceedings of the ASME 2014 33rd International Conference on Ocean, Offshore and Arctic Engineering, OMAE2014, Sa n Francisco, CA, USA, 8–13 June 2014.
- 70. API. API 17K: Specification for Bonded Flexible Pipe, 3rd ed.; American Petroleum Institute (API): Washington, DC, US A, 2017.

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